

Numerical Weather Prediction Parametrization of diabatic processes

Convection II: The parametrization

Peter Bechtold, Christian Jakob, David Gregory
(with contributions from J. Kain (NOAA/NSLL))



Outline

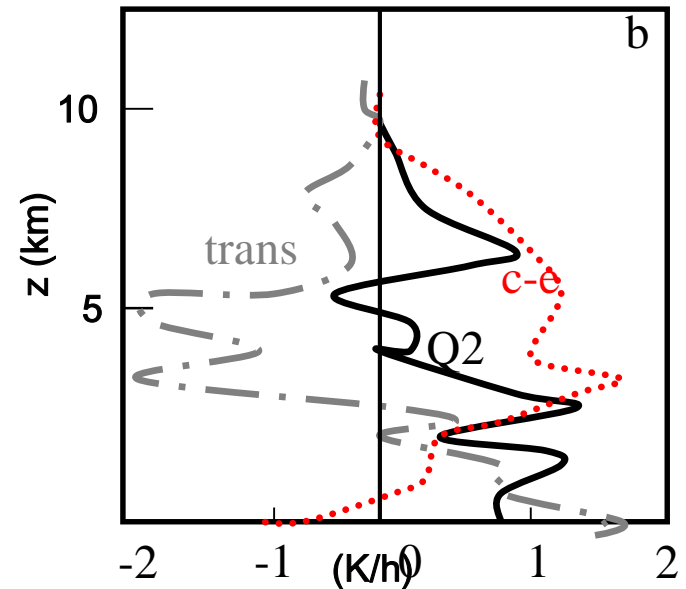
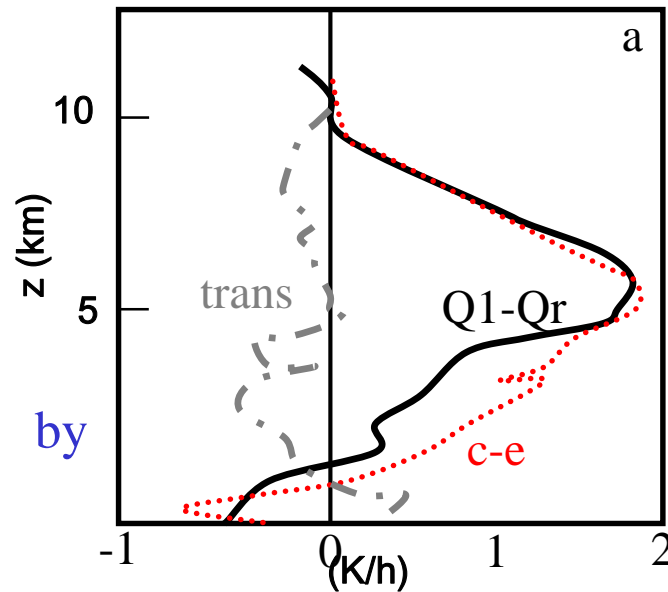
- Aims of convection parametrization
- Overview over approaches to convection parametrization
- The mass-flux approach

Task of convection parametrisation

total Q1 and Q2

To calculate the collective effects of an ensemble of convective clouds in a model column as a function of grid-scale variables. Hence parameterization needs to describe Condensation/Evaporation and Transport

$$Q_{1c} \equiv Q_1 - Q_R \equiv L(\bar{c} - \bar{e}) - \frac{\overline{\partial \omega' s'}}{\partial p}$$



Q1c is dominated by condensation term

but for Q2 the transport and condensation terms are equally important

Caniaux, Redelsperger, Lafore, JAS 1994

Task of convection parametrisation in practice this means:

Determine **occurrence/localisation** of convection

—————→ **Trigger**

Determine **vertical distribution** of heating, moistening and momentum changes

—————→ **Cloud model**

Determine the **overall amount** of the energy conversion, convective precipitation=heat release

—————→ **Closure**

Constraints for convection parametrization

- Physical
 - remove **convective instability** and produce **subgrid-scale convective precipitation (heating/drying)** in unsaturated model grids
 - produce a **realistic mean tropical climate, coupling with microphysics**
 - maintain a **realistic variability** on a wide range of time-scales
 - be applicable to a **wide range of scales** (typical 5 - 200 km) and **types of convection** (deep tropical, shallow, midlatitude and front/post-frontal convection)
- Computational
 - be **simple and efficient** for different model/forecast configurations T1279 (10-16 km), EPS, seasonal prediction T255 (80 km))

Types of convection schemes

- Schemes based on **moisture budgets**
 - Kuo, 1965, 1974, *J. Atmos. Sci.*
- **Adjustment** schemes
 - moist convective adjustment, Manabe, 1965, *Mon. Wea. Rev.*
 - penetrative adjustment scheme, Betts and Miller, 1986, *Quart. J. Roy. Met. Soc.*, Betts-Miller-Janic
- **Mass-flux** schemes (bulk+spectral)
 - entraining plume - spectral model, Arakawa and Schubert, 1974, Fraedrich (1973,1976), Neggers et al (2002), Cheinet (2004), *all J. Atmos. Sci.* ,
 - Entraining/detraining plume - bulk model, e.g., Bougeault, 1985, *Mon. Wea. Rev.*, Tiedtke, 1989, *Mon. Wea. Rev.*; Gregory and Rowntree, 1990, *Mon. Wea. Rev.*; Kain and Fritsch, 1990, *J. Atmos. Sci.*, Donner , 1993 *J. Atmos. Sci.*; Bechtold et al 2001, *Quart. J. Roy. Met. Soc.*; Park, 2014. *J. Atmos. Sci.*
 - episodic mixing, Emanuel, 1991, *J. Atmos. Sci.*

The “Kuo” scheme closure

Closure: Convective activity is linked to large-scale moisture convergence

$$P = (1 - b) \int_0^{\infty} \left(\frac{\partial \rho q}{\partial t} \right)_{ls} dz$$

Vertical distribution of heating and moistening: adjust grid-mean to moist adiabat

Main problem: here convection is assumed to consume water and not energy -> Positive feedback loop of moisture convergence, people criticized but also works

Adjustment schemes

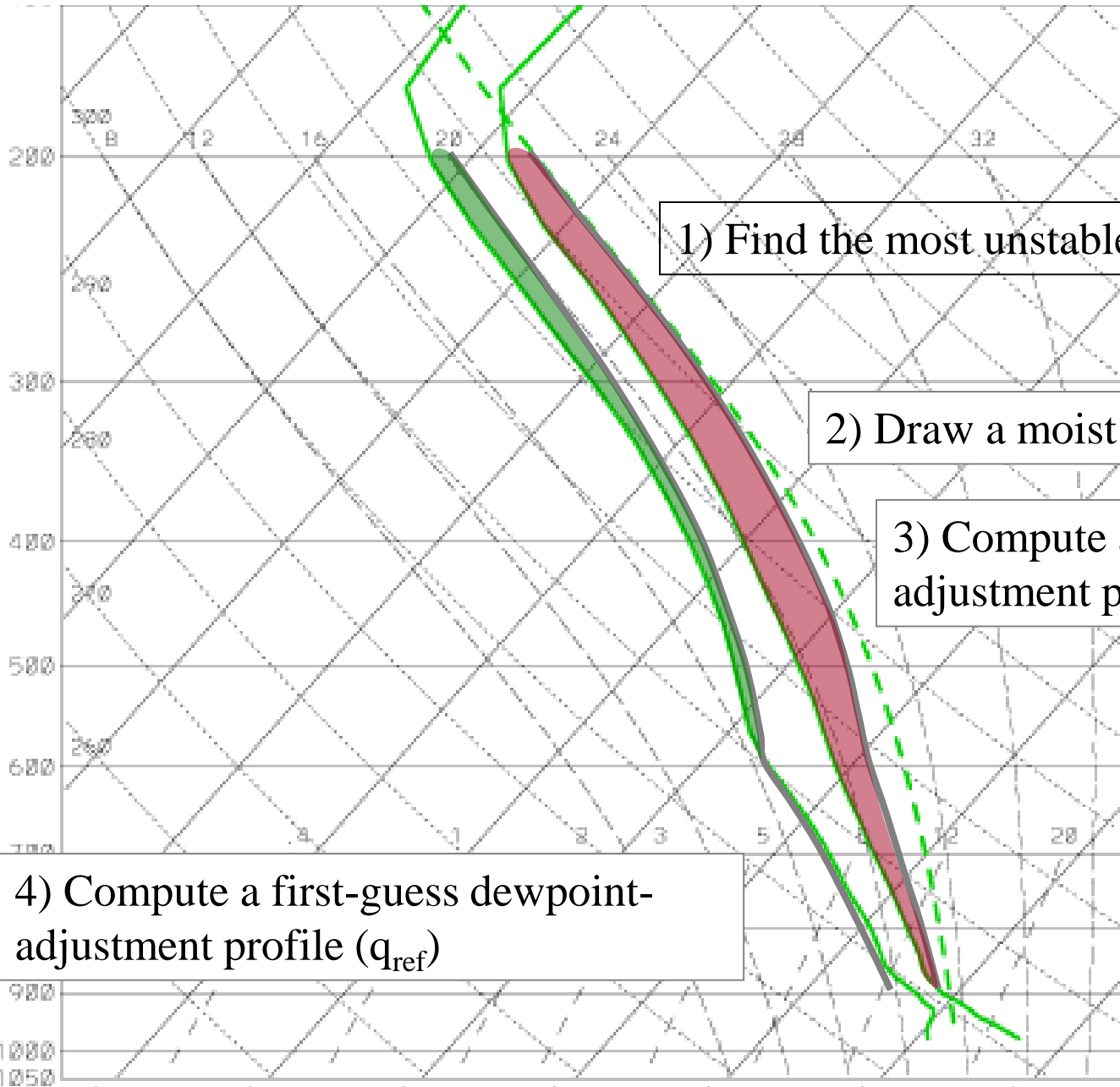
e.g. Betts and Miller, 1986, QJRMS:

When **atmosphere is unstable to parcel lifted from PBL** and there is a deep moist layer - **adjust state back to reference profile** over some time-scale, i.e.,

$$\left(\frac{\partial T}{\partial t}\right)_{conv.} = \frac{T_{ref} - T}{\tau} \quad \left(\frac{\partial q}{\partial t}\right)_{conv.} = \frac{q_{ref} - q}{\tau}$$

T_{ref} is constructed from moist adiabat from cloud base but **no universal reference profiles** for q exist. However, scheme is robust and produces “smooth” fields.

Procedure followed by Betts Miller Janjić scheme...



1) Find the most unstable air in lowest ~ 200 mb

2) Draw a moist adiabat for this air

3) Compute a first-guess temperature-adjustment profile (T_{ref})

4) Compute a first-guess dewpoint-adjustment profile (q_{ref})

Adjustment schemes:

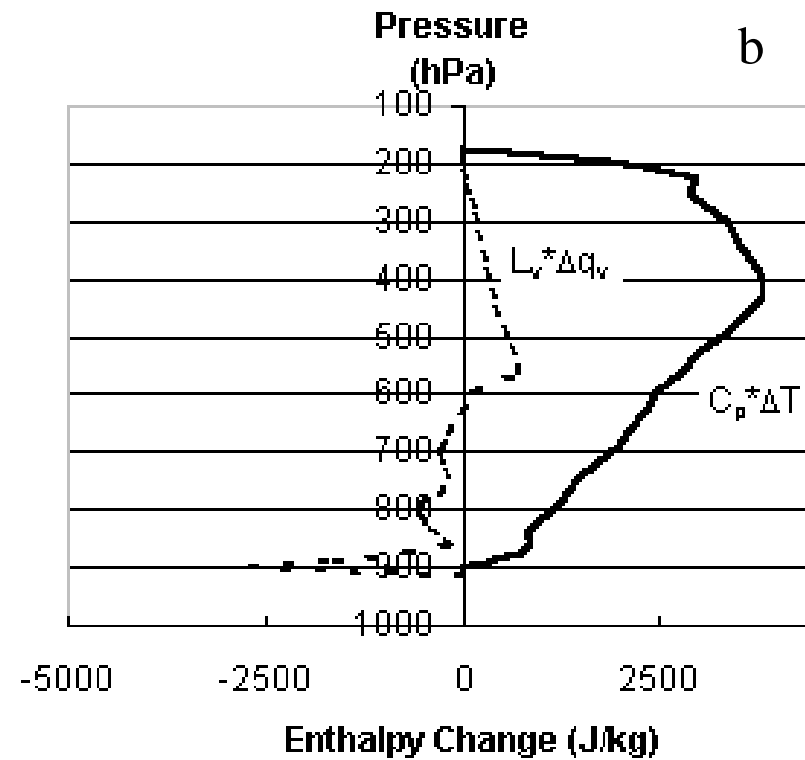
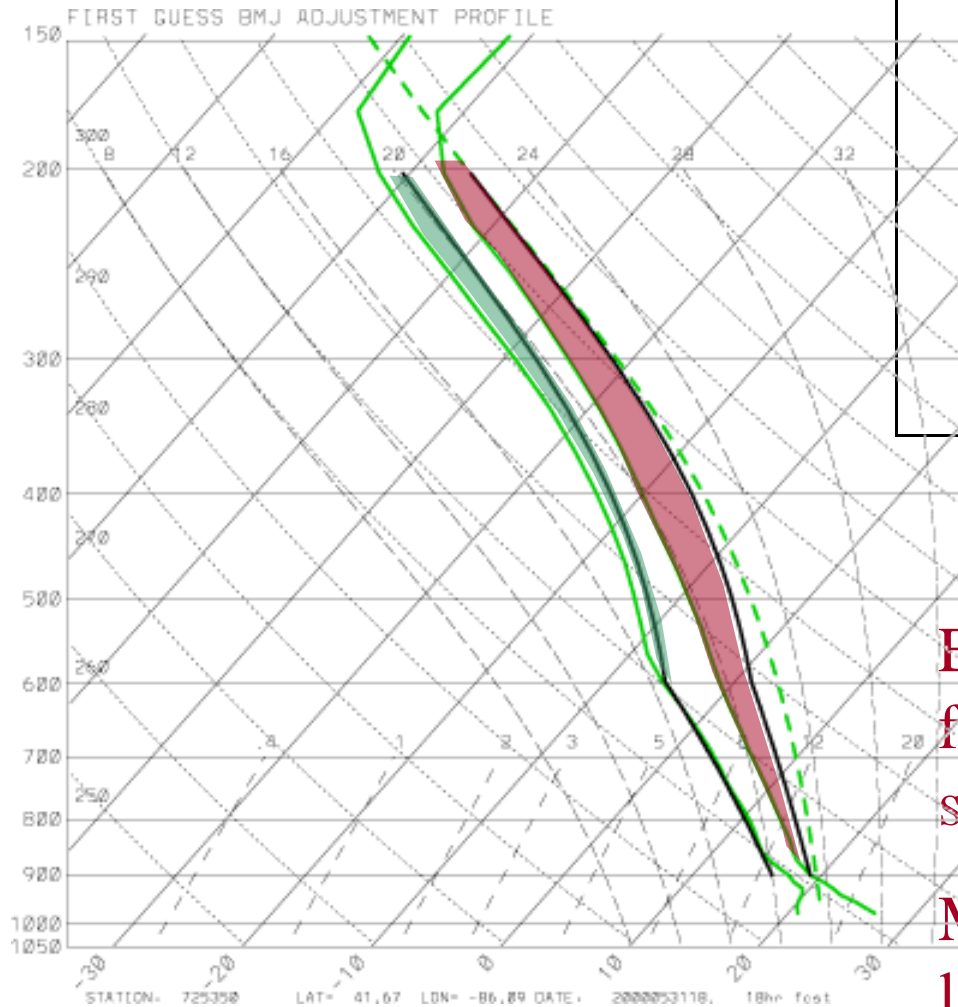
The Next Step is an *Enthalpy* Adjustment

First Law of
Thermodynamics:

$$dH = C_p dT + L_v dq_v$$

With Parameterized Convection, each grid-point column is treated in isolation. Total column latent heating must be directly proportional to total column drying, or $dH = 0$.

$$\int_{P_b}^{P_t} C_p (T_{ref} - T) dp = - \int_{P_b}^{P_t} L_v (q_{vref} - q_v) dp$$



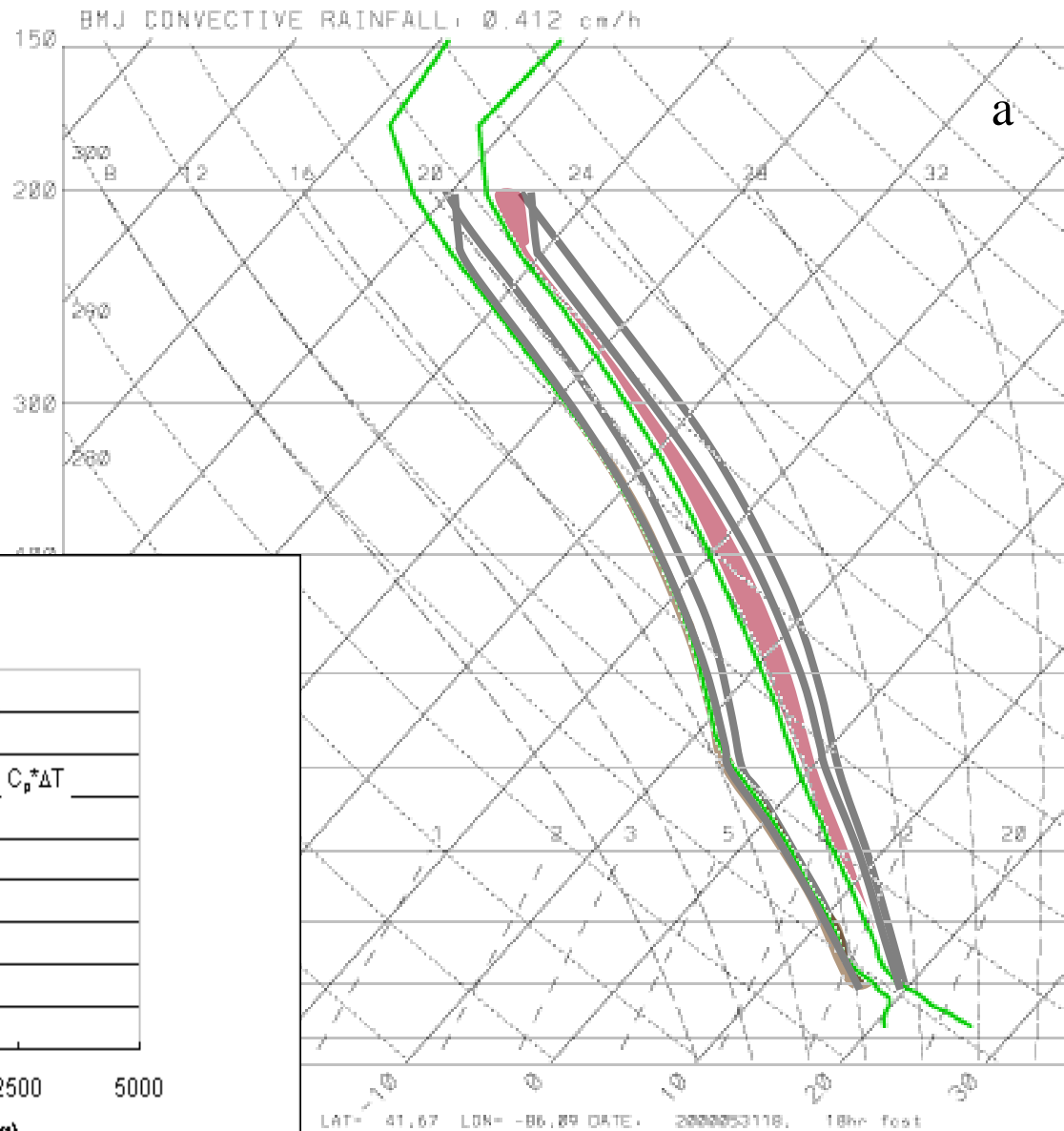
Enthalpy is *not* conserved for first-guess profiles for this sounding!

Must shift T_{ref} and q_{vref} to the left...

Imposing Enthalpy Adjustment:

b

Shift profiles to the left in order to conserve enthalpy



The mass-flux approach

$$Q_{1c} \equiv L(\bar{c} - \bar{e}) - \frac{\overline{\partial \omega' s'}}{\partial p}$$

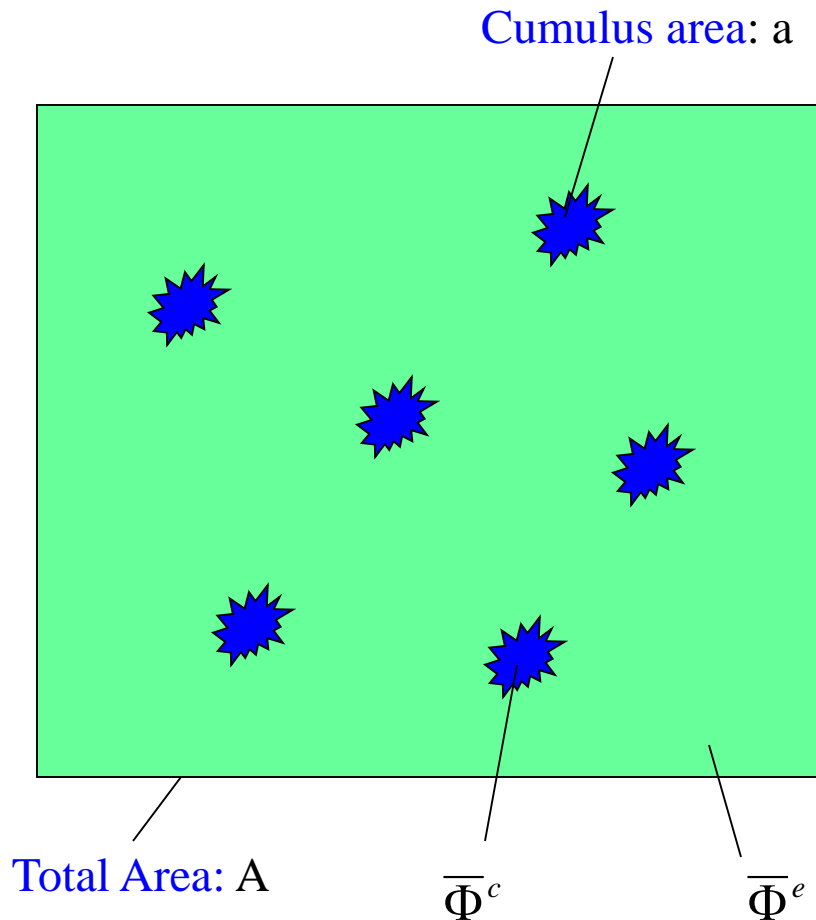
Condensation term

Eddy transport term

Aim: Look for a simple expression of the eddy transport term

$$\overline{\omega' \Phi'} = ?$$

The mass-flux approach: Cloud – Environment decomposition



Fractional coverage with
cumulus elements:

$$\sigma = \frac{a}{A}$$

Define area average:

$$\bar{\Phi} = \sigma \bar{\Phi}^c + (1 - \sigma) \bar{\Phi}^e$$

The mass-flux approach: Cloud-Environment decomposition

With the above:

$$\overline{\omega\Phi} = \sigma \overline{\omega\Phi}^c + (1 - \sigma) \overline{\omega\Phi}^e$$

Average over cumulus elements

Average over environment

$$\overline{\omega\Phi}^c = \bar{\omega}^c \bar{\Phi}^c + \cancel{\overline{\omega''\Phi''}^c} \quad \text{and} \quad \overline{\omega\Phi}^e = \bar{\omega}^e \bar{\Phi}^e + \cancel{\overline{\omega''\Phi''}^e}$$

Neglect subplume variations : (1) The top hat assumption

(see also Siebesma and Cuijpers, JAS 1995 for a discussion of the validity of the top-hat assumption)

The mass-flux approach:

Derivation 1

(my preferred)

$$\overline{\omega'\Phi'} = \overline{\omega\Phi} - \bar{\omega}\bar{\Phi} = \sigma\overline{\omega\Phi^c} + (1-\sigma)\overline{\omega\Phi^e} - \bar{\omega}\bar{\Phi}$$

Use Reynolds averaging again for cumulus elements and environment separately:

$$= \sigma\overline{\omega\Phi^c} + (1-\sigma)\overline{\omega\Phi^e} - (\sigma\bar{\omega}^c + (1-\sigma)\bar{\omega}^e)\bar{\Phi}$$

top hat

$$= \sigma\bar{\omega}^c\bar{\Phi}^c + (1-\sigma)\bar{\omega}^e\bar{\Phi}^e - (\sigma\bar{\omega}^c + (1-\sigma)\bar{\omega}^e)\bar{\Phi}$$

small area

$$= \sigma\bar{\omega}^c(\bar{\Phi}^c - \bar{\Phi}) + (1-\sigma)\bar{\omega}^e(\bar{\Phi}^e - \bar{\Phi})$$

Further simplifications : (2) The small area approximation

$$\sigma \ll 1 \Rightarrow (1 - \sigma) \approx 1; \quad \bar{\omega}^c \gg \bar{\omega}^e$$

The mass-flux approach: Derivation 2

Then after some algebra (for your exercise) :

$$\begin{aligned}\overline{\omega'\Phi'} &= \overline{\omega\Phi} - \overline{\omega}\overline{\Phi} \\ &= \sigma(1-\sigma)(\overline{\omega}^c - \overline{\omega}^e)(\overline{\Phi}^c - \overline{\Phi}^e)\end{aligned}$$

Further simplifications :

The small area approximation

$$\sigma \ll 1 \Rightarrow (1 - \sigma) \approx 1; \quad \overline{\omega}^c \gg \overline{\omega}^e$$

The mass-flux approach

Then :

$$\overline{\omega'\Phi'} = \sigma\bar{\omega}^c (\bar{\Phi}^c - \bar{\Phi}^e)$$

Define convective mass-flux:

$$M_c = \frac{-\sigma\bar{\omega}^c}{g} = \rho\sigma\bar{w}^c$$

Then

$$-\overline{\omega'\Phi'} = gM_c (\bar{\Phi}^c - \bar{\Phi})$$

The mass-flux approach

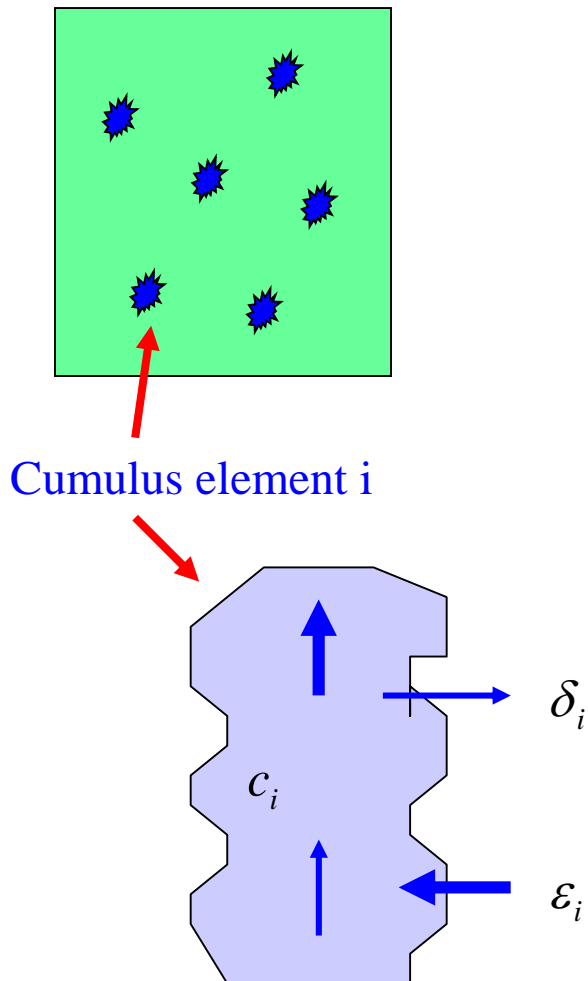
With the above we can rewrite:

$$Q_{1c} \equiv L(\bar{c} - \bar{e}) + g \frac{\partial [M_c (\bar{s}^c - \bar{s})]}{\partial p}$$

$$Q_2 \equiv L(\bar{c} - \bar{e}) - Lg \frac{\partial [M_c (\bar{q}^c - \bar{q})]}{\partial p}$$

To predict the **influence of convection** on the large-scale we now need to describe **the convective mass-flux, the values (s, q, u, v) inside the convective elements and the condensation/evaporation term**. This requires, as usual, a **cloud model** and a **closure** to determine the absolute (scaled) value of the mass flux.

Mass-flux entraining plume models



Entraining plume model

Continuity:

$$\frac{\partial \sigma_i}{\partial t} + D_i - E_i - g \frac{\partial M_i}{\partial p} = 0$$

Heat:

$$\frac{\partial (\sigma_i s_i)}{\partial t} + D_i s_i - E_i \bar{s} - g \frac{\partial (M_i s_i)}{\partial p} = L c_i$$

Specific humidity:

$$\frac{\partial (\sigma_i q_i)}{\partial t} + D_i q_i - E_i \bar{q} - g \frac{\partial (M_i q_i)}{\partial p} = -c_i$$

Mass-flux entraining plume models

Simplifications

1. **Steady state plumes**, i.e., $\frac{\partial X}{\partial t} = 0$

most mass-flux convection parametrizations make that assumption, some (e.g. Gerard&Geleyn) are prognostic

2. Instead of spectral (Arakawa Schubert 1974) use **one representative updraught=bulk scheme** with entrainment/detrainment written as

$$\frac{1}{M} \frac{dM}{dz} = \varepsilon - \delta \Rightarrow -g \frac{\partial M_c}{\partial p} = E - D$$

ε, δ [m^{-1}] denote fractional entrainment/detrainment,
 E, D [s^{-1}] entrainment/detrainment rates

Large-scale cumulus effects deduced from mass-flux models

$$-g \frac{\partial M_c}{\partial p} = E - D$$

$$-g \frac{\partial (M_c \bar{s}^c)}{\partial p} = E\bar{s} - D\bar{s}^c + Lc$$

$$Q_{1c} \equiv L(c - e) + g \frac{\partial [M_c (\bar{s}^c - \bar{s})]}{\partial p}$$

Flux form

Combine:

$$Q_{1c} \equiv -gM_c \frac{\partial \bar{s}}{\partial p} + D(\bar{s}^c - \bar{s}) - Le$$

Advective form

Large-scale cumulus effects deduced using mass-flux models

$$Q_{1c} \equiv -gM_c \frac{\partial \bar{s}}{\partial p} + D(\bar{s}^c - \bar{s}) - Le$$

Physical interpretation :

Convection affects the large scales by

Heating through **compensating subsidence** between cumulus elements (term 1)

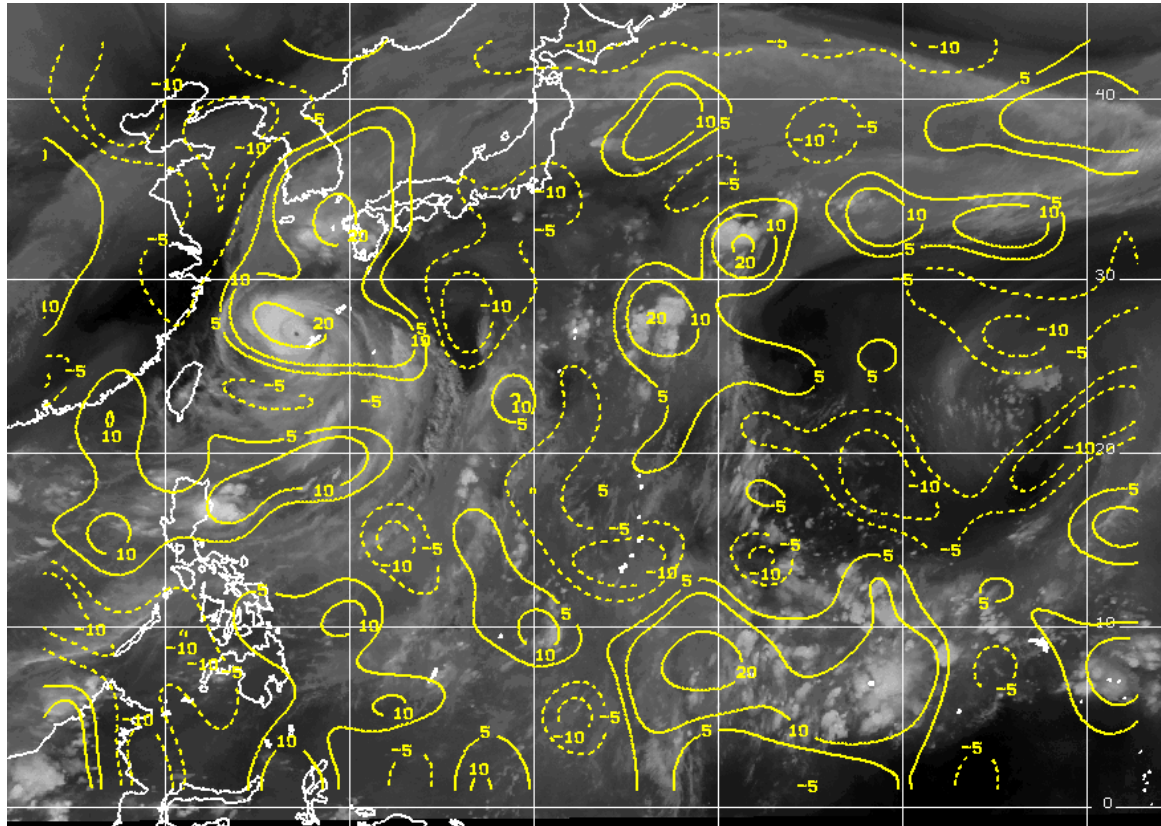
The **detrainment of cloud air** into the environment (term 2)

Evaporation of cloud and precipitation (term 3)

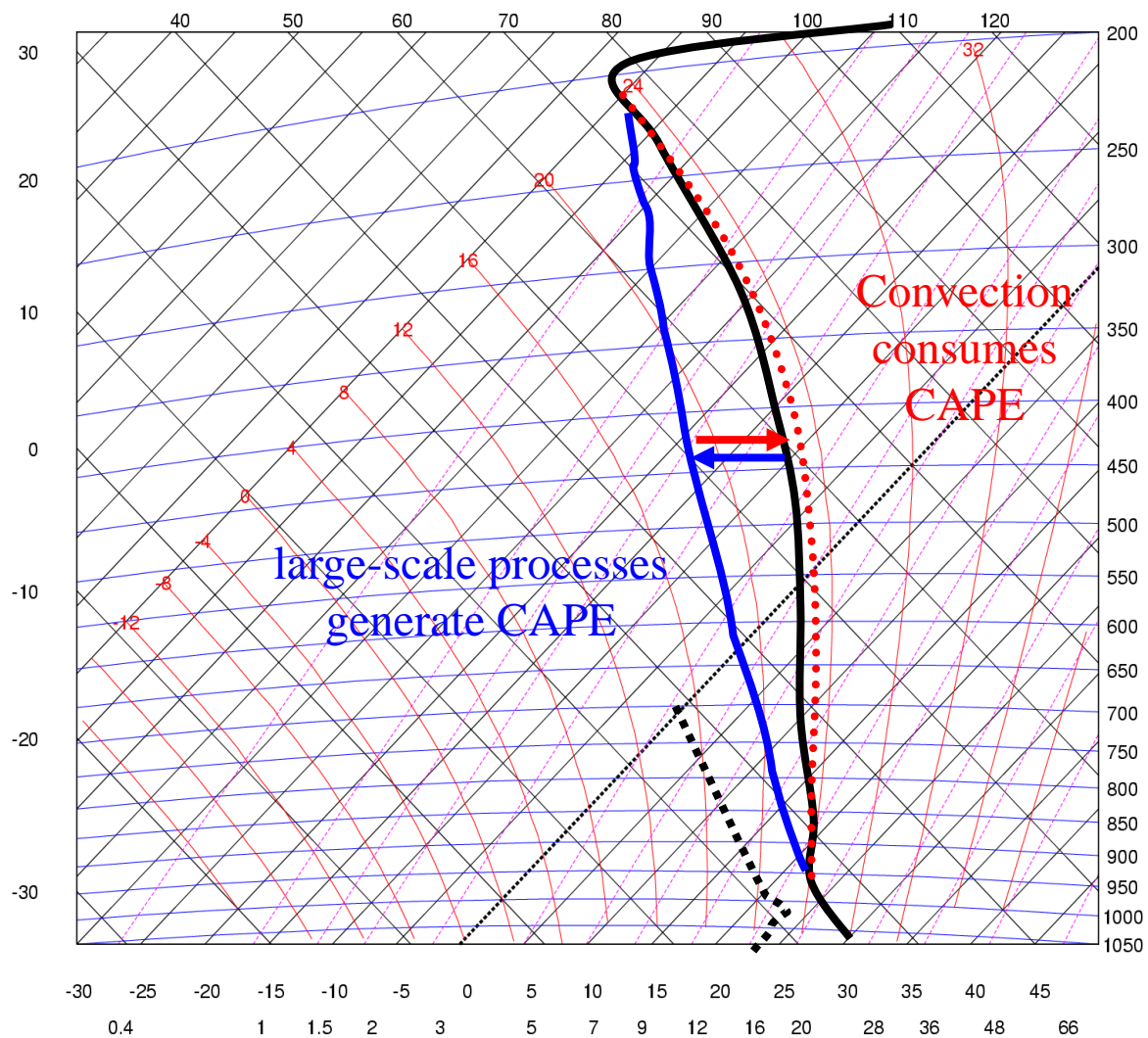
Note: In the **advective form** the **condensation heating** does **not** appear **directly** in Q_1 . It is however the dominant term using the **flux form** and is a **crucial part of the cloud model**, where this heat is transformed in kinetic energy of the updrafts.

Determine mass flux change= divergence from variation of cold cloud top areas

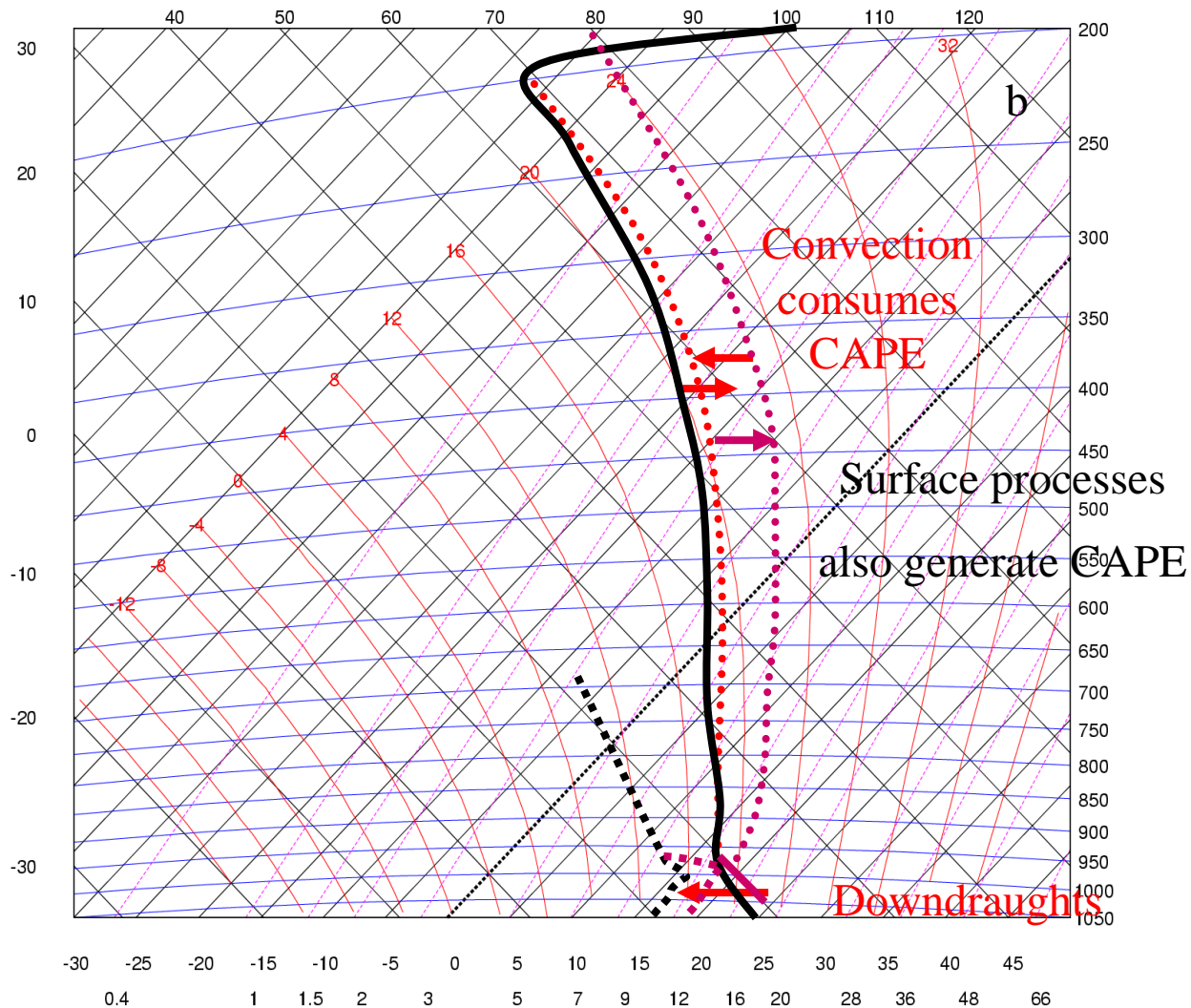
$$\text{Div}\rho\vec{U} = \lim_{\text{Vol}\rightarrow 0} \frac{1}{\text{Vol}} \iint_A \rho\vec{U}d\vec{A} = \lim_{\Delta t\rightarrow 0} \frac{1}{\text{Vol}} \frac{\Delta m}{\Delta t} \approx \rho \frac{1}{A} \frac{dA}{dt} = \frac{\partial M_c}{\partial z}$$



CAPE closure - the basic idea



CAPE closure - the basic idea



Summary

- **Convection parametrisations** need to provide a **physically realistic** forcing/response on the resolved model scales and need to be **practical**
- a **number of approaches** to convection parametrisation exist
- **basic ingredients** to present convection parametrisations are a method to **trigger convection**, a **cloud model** and a **closure assumption**
- the **mass-flux approach** has been successfully applied to both interpretation of data and convection parametrization

More Tricks and Tips for mass flux parametrization in Lecture III on the IFS scheme